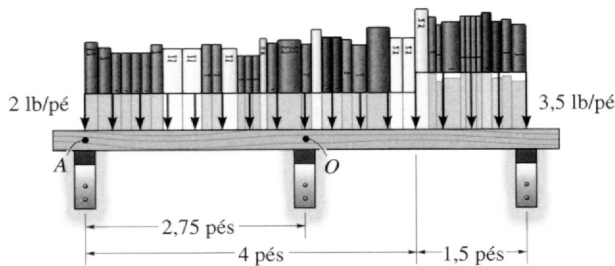


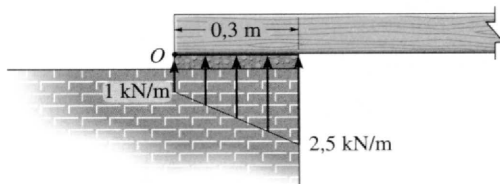
## PROBLEMAS

**4.139.** As cargas na estante de livros estão distribuídas como mostrado na figura. Determine a intensidade da força resultante equivalente e sua localização, tomando como origem o ponto  $O$ .



**Problema 4.139**

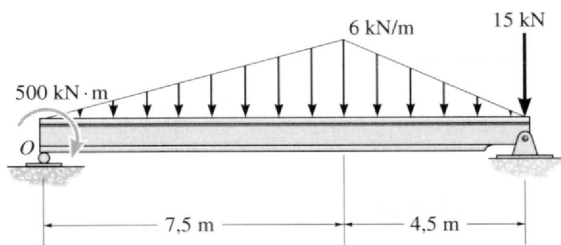
**\*4.140.** O suporte de alvenaria gera a distribuição de cargas atuando nas extremidades da viga. Simplifique essas cargas a uma única força resultante e especifique sua localização, medida a partir do ponto  $O$ .



**Problema 4.140**

**4.141.** Substitua as cargas por uma força e um momento equivalentes, atuantes no ponto  $O$ .

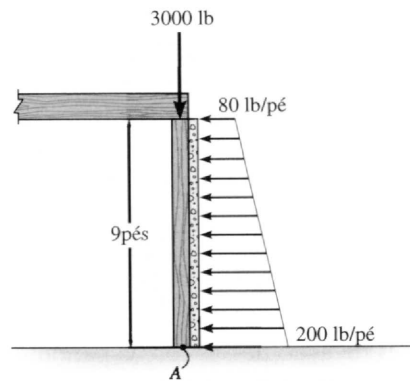
**4.142.** Substitua as cargas por uma única força resultante e especifique a localização dessa força sobre a viga, medida a partir do ponto  $O$ .



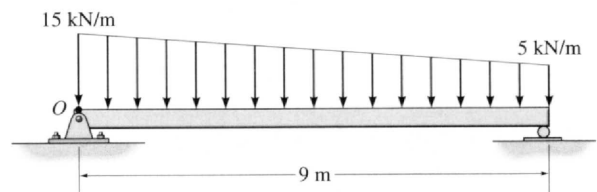
**Problemas 4.141/142**

**4.143.** A coluna é usada para sustentar o piso superior, que exerce uma força de 3.000 lb no topo dela. O efeito da pressão do solo na lateral da coluna é distribuído como mostrado na figura. Substitua esse carregamento por uma força resultante equivalente e especifique em que ponto a força atua ao longo da coluna, a partir de sua base  $A$ .

**\*4.144.** Substitua as cargas por uma força e um momento equivalentes que atuam no ponto  $O$ .

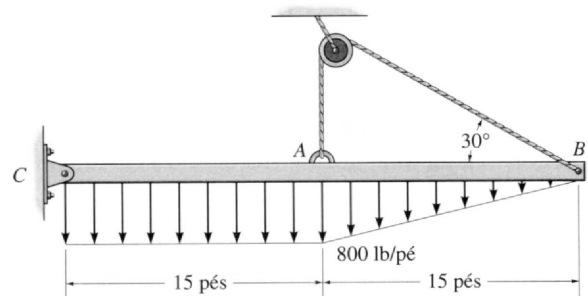


**Problema 4.143**



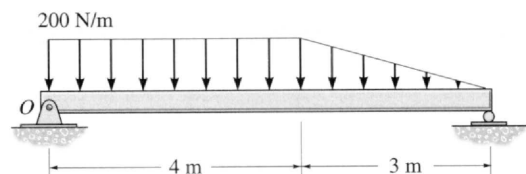
**Problema 4.144**

**4.145.** Substitua o carregamento distribuído por uma força resultante equivalente e especifique sua localização sobre a viga, a partir do pino em  $C$ .



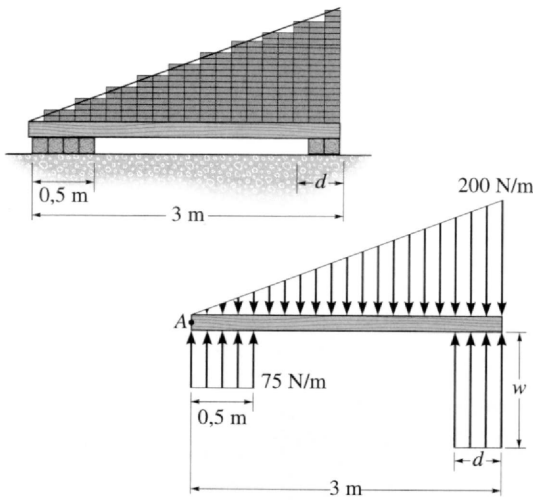
**Problema 4.145**

**4.146.** Substitua as cargas por uma força e um momento equivalentes, atuantes no ponto  $O$ .



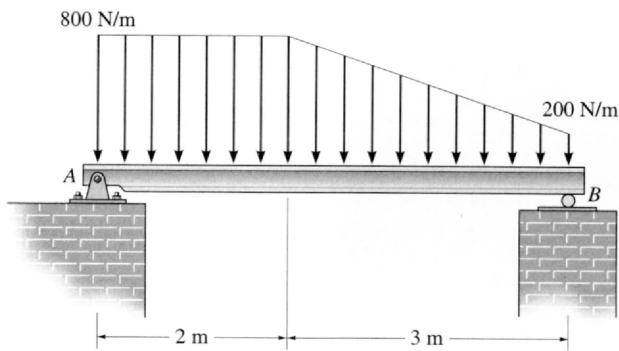
**Problema 4.146**

**4.147.** Os tijolos sobre a viga e os suportes sobre o solo geram o carregamento distribuído mostrado na segunda figura. Determine a intensidade necessária  $w$  e a dimensão  $d$  do suporte direito de modo que a força resultante e o momento em relação ao ponto  $A$  do sistema sejam ambos nulos.



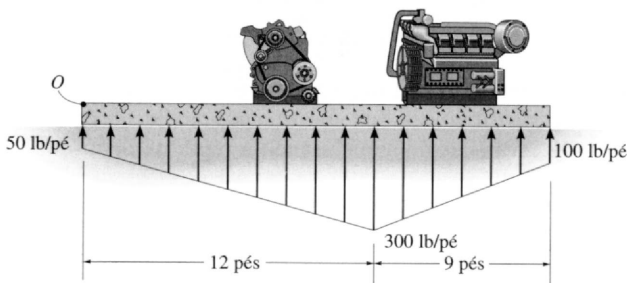
Problema 4.147

\*4.148. Substitua o carregamento distribuído por uma força resultante equivalente e especifique sua localização medida a partir do ponto *A*.



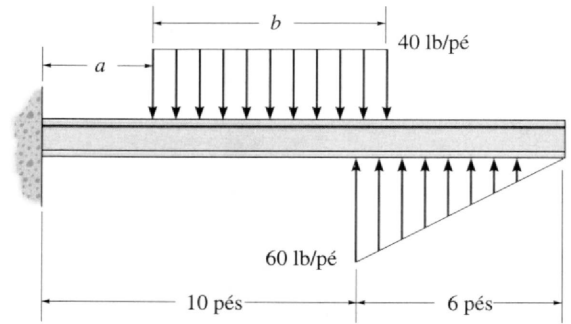
Problema 4.148

4.149. A distribuição das cargas do solo na base de uma laje é mostrada na figura. Substitua essas cargas por uma força resultante equivalente e especifique sua localização, tomando como referência o ponto *O*.



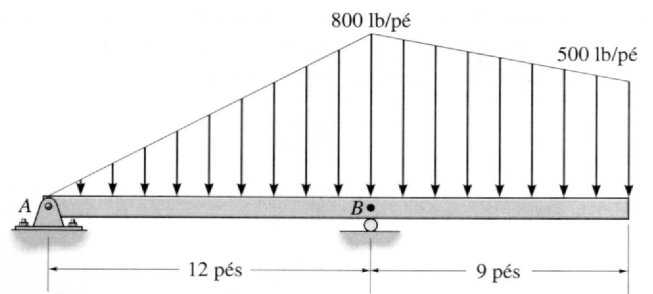
Problema 4.149

4.150. A viga é submetida ao carregamento distribuído mostrado na figura. Determine o comprimento *b* da distribuição uniforme de cargas e o posicionamento *a* para que a força resultante e o momento atuantes na viga sejam ambos nulos.



Problema 4.150

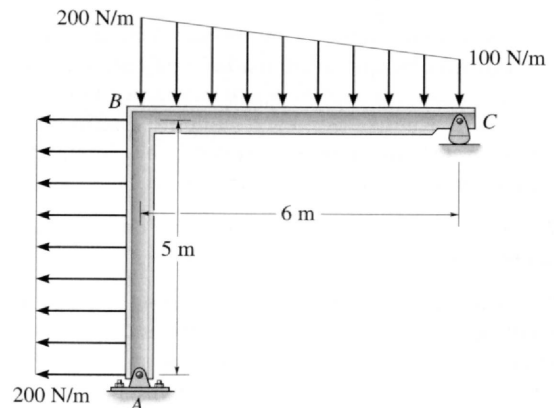
4.151. Substitua as cargas por uma força resultante equivalente e especifique sua localização sobre a viga, medida a partir do ponto *B*.



Problema 4.151

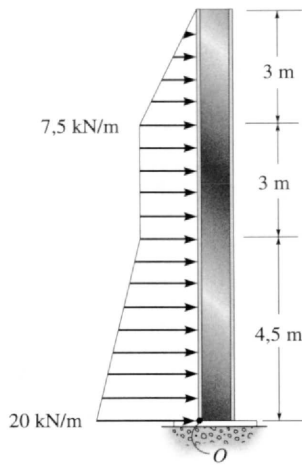
\*4.152. Substitua as cargas distribuídas por uma força resultante equivalente e especifique onde sua linha de ação intercepta o elemento *AB*, medido a partir de *A*.

4.153. Substitua as cargas distribuídas por uma força resultante equivalente e especifique onde sua linha de ação intercepta o elemento *BC*, medido a partir de *C*.



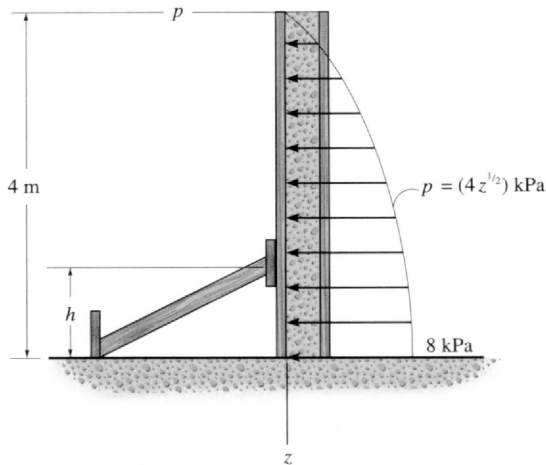
Problemas 4.152/153

4.154. Substitua as cargas por uma força resultante e momento equivalentes atuantes no ponto *O*.



Problema 4.154

**4.155.** O concreto molhado exerce uma pressão distribuída ao longo das paredes da fôrma. Determine a força resultante dessa distribuição e especifique a altura  $h$  em que a escora deve ser colocada para que se posicione na linha de ação da força resultante. O muro tem espessura de 5 m.



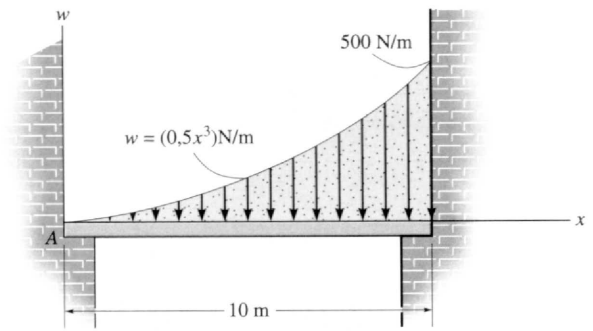
Problema 4.155

**\*4.156.** A ação do vento criou um depósito de areia sobre uma plataforma tal que a intensidade da carga de areia sobre essa plataforma pode ser aproximada por uma função  $w = (0,5x^3)$  N/m. Reduza esse carregamento distribuído a uma força resultante equivalente e especifique a intensidade e a localização da força, medida a partir de A.

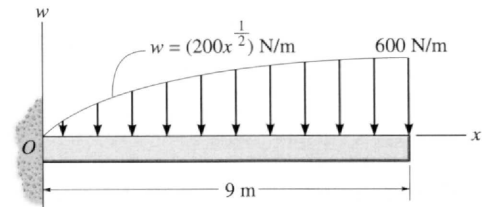
**4.157.** Substitua o carregamento por uma força e um momento equivalentes atuantes no ponto O.

**4.158.** A força de sustentação ao longo da asa de um avião a jato consiste em uma distribuição uniforme ao longo da distância AB e uma distribuição parabólica no trecho BC, com origem em B. Substitua esse carregamento por uma única força resultante e especifique sua localização, medida a partir do ponto A.

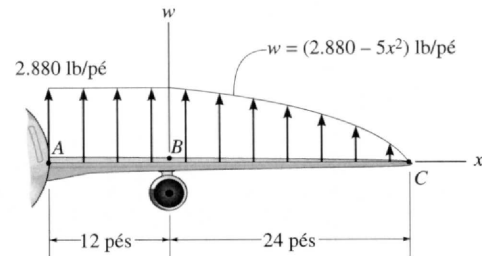
**4.159.** Determine a intensidade da força resultante equivalente da distribuição de cargas e especifique sua localização sobre a viga, medida a partir do ponto A.



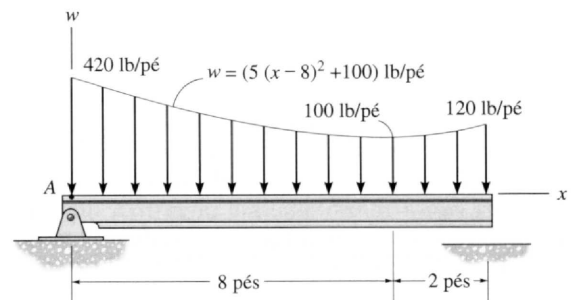
Problema 4.156



Problema 4.157

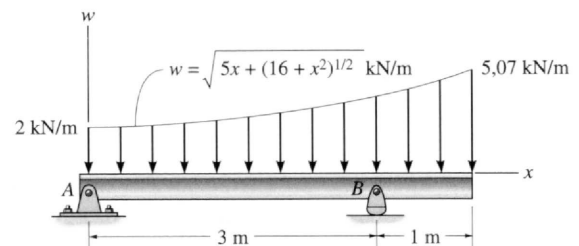


Problema 4.158



Problema 4.159

**\*4.160.** Determine a força resultante equivalente do carregamento distribuído e sua localização, medida a partir do ponto A. Avalie as integrais usando a regra de Simpson.



Problema 1.160

## REVISÃO DO CAPÍTULO

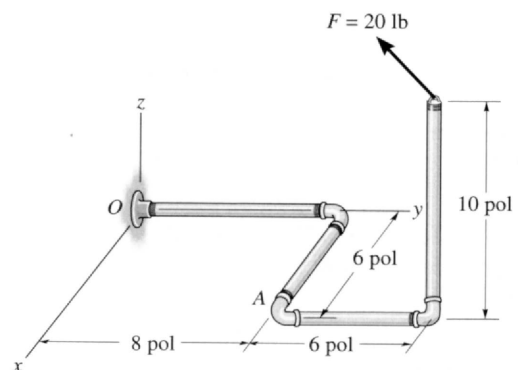
- Momento de uma Força.** Uma força produz um efeito de giro em torno de um ponto  $O$  que não se localiza em sua linha de ação. Na forma escalar, a *intensidade* do momento é  $M_O = Fd$ , onde  $d$  é o braço do momento ou distância perpendicular do ponto  $O$  à linha de ação da força. A *direção* e o *sentido* do momento são definidos utilizando a regra da mão direita. Em vez de encontrar  $d$ , costuma ser mais fácil decompor a força em seus componentes  $x$  e  $y$ , determinar o momento de cada componente em relação ao ponto e somar os resultados vetorialmente. Uma vez que a geometria tridimensional em geral é mais difícil de visualizar, o produto vetorial pode ser usado para determinar o momento  $\mathbf{M}_O = \mathbf{r} \times \mathbf{F}$ , onde  $\mathbf{r}$  é o vetor posição que se estende desde o ponto  $O$  até qualquer ponto sobre a linha de ação de  $\mathbf{F}$ .
- Momento em Relação a um Eixo Específico.** Se o momento de uma força em relação a um eixo arbitrário deve ser determinado, é necessário obter a projeção do momento sobre o eixo. Contanto que a distância  $d_a$ , que é perpendicular *tanto* à linha de ação da força *quanto* ao eixo, possa ser determinada, o momento da força em relação ao eixo é simplesmente  $M_a = Fd_a$ . Se a distância  $d_a$  não pode ser encontrada, então o produto vetorial tríplice deve ser usado, sendo  $M_a = \mathbf{u}_a \cdot \mathbf{r} \times \mathbf{F}$ . Nesse caso,  $\mathbf{u}_a$  é o vetor unitário que especifica a direção do eixo e  $\mathbf{r}$  é um vetor de posição que está orientado de um ponto arbitrário sobre o eixo até qualquer ponto da linha de ação da força.
- Momento de Binário.** Um binário consiste de duas forças iguais em intensidade, porém de sentidos opostos, que atuam a uma distância perpendicular  $d$  entre elas. Os binários tendem a produzir rotação sem translação. O momento de um binário é determinado de  $M = Fd$  e sua direção e seu sentido são estabelecidos por meio da regra da mão direita. Se o produto vetorial for usado para determinar o momento do binário, então  $\mathbf{M} = \mathbf{r} \times \mathbf{F}$ . Nesse caso,  $\mathbf{r}$  tem origem em qualquer ponto sobre a linha de ação de uma das forças e extremidade em qualquer ponto sobre a linha de ação da força  $\mathbf{F}$  utilizada no produto vetorial.
- Redução de um Sistema de Forças e Momentos.** Qualquer sistema de forças e momentos pode ser reduzido a uma única força resultante e a um momento resultante atuando em um ponto. A força resultante é a soma vetorial de todas as forças do sistema e o momento resultante é igual ao momento da força resultante adicionado aos momentos do sistema no mesmo ponto. Simplificações adicionais para uma única força resultante são possíveis se o sistema de forças é *concorrente*, *coplanar* ou *paralelo*. Para encontrar a localização da força resultante em relação a um ponto, nesses casos, é necessário igualar o momento da força resultante aos momentos das forças e momentos do sistema em relação ao mesmo ponto. Com esse procedimento, para qualquer outro *tipo* de sistema de forças gerará um *torsor*, que consiste em uma força resultante e momento colineares.
- Carregamento Distribuído.** Um carregamento distribuído simples pode ser substituído por uma *força resultante*, que é equivalente à *área* sob a curva de carregamento. Essa resultante tem uma linha de ação que passa pelo *centróide* ou centro geométrico da área ou volume sob a curva do diagrama de carregamento.

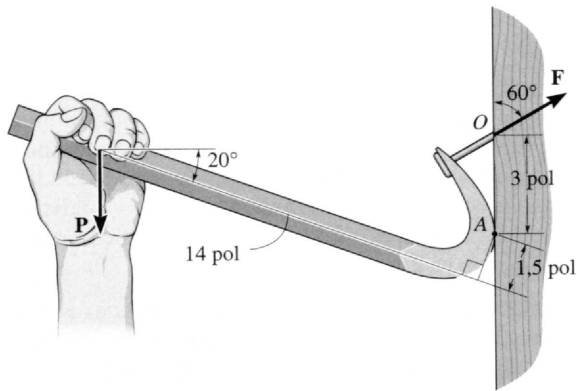
## PROBLEMAS DE REVISÃO

**4.161.** Determine os ângulos diretores coordenados  $\alpha$ ,  $\beta$  e  $\gamma$  de  $\mathbf{F}$ , que é aplicado à extremidade  $A$  de uma estrutura tubular, de modo que o momento de  $\mathbf{F}$  em relação a  $O$  seja nulo.

**4.162.** Determine o momento da força  $\mathbf{F}$  em relação ao ponto  $O$ . A força tem ângulos diretores coordenados  $\alpha = 60^\circ$ ,  $\beta = 120^\circ$  e  $\gamma = 45^\circ$ . Expresse o resultado como um vetor cartesiano.

**4.163.** Se uma força  $F = 125$  lb é utilizada para retirar um prego, determine a menor força vertical  $\mathbf{P}$  que deve ser aplicada ao cabo do pé-de-cabra. *Dica:* os momentos de  $\mathbf{F}$  e de  $\mathbf{P}$  em relação ao ponto  $A$  devem ser iguais. Por quê?

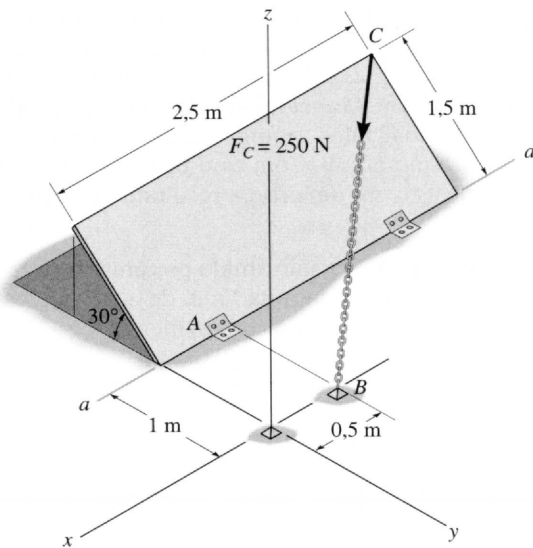




**Problema 4.163**

**\*4.164.** Determine o momento da força  $F_c$  em relação à dobradiça A da porta. Expresse o resultado como um vetor cartesiano.

**4.165.** Determine a intensidade do momento de força  $F_c$  em relação ao eixo  $aa$  das dobradiças da porta.

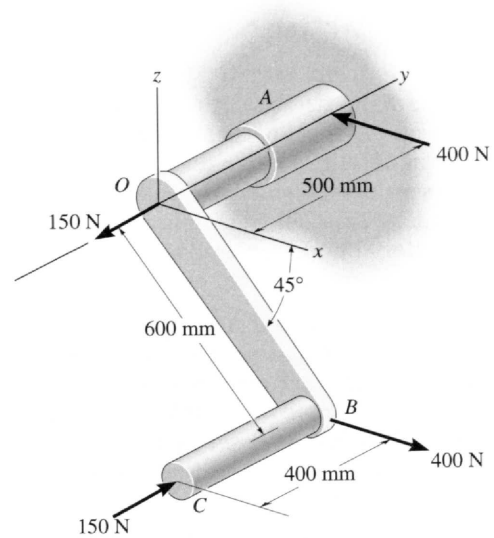


**Problemas 4.164/165**

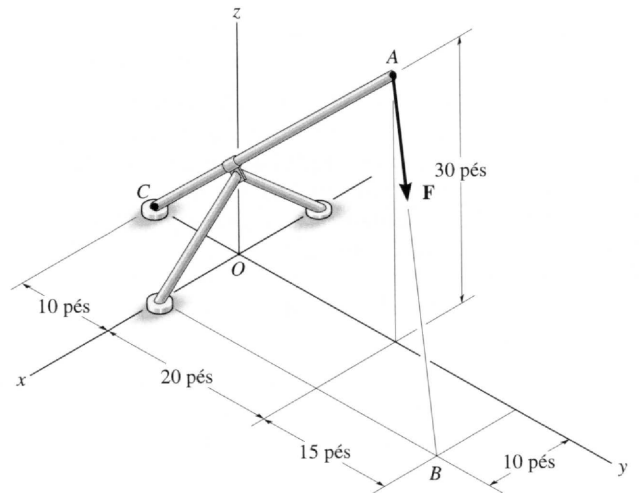
**4.166.** Determine o momento resultante dos dois binários que atuam na estrutura. O elemento  $OB$  se encontra no plano  $x-y$ .

**4.167.** Substitua a força  $F$  que tem intensidade  $F = 50$  lb e atua no ponto  $A$  por uma força e um momento equivalentes no ponto  $C$ .

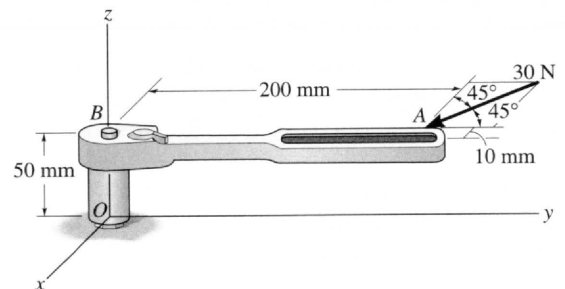
**\*4.168.** A força horizontal de 30 N é aplicada ao cabo da chave. Qual é a intensidade do momento dessa força em relação ao eixo  $z$ ?



**Problema 4.166**

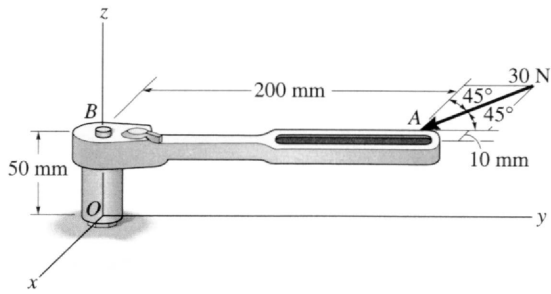


**Problema 4.167**



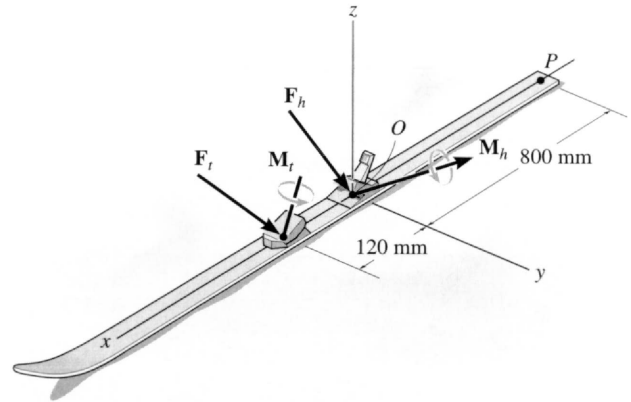
**Problema 4.168**

**4.169.** A força horizontal de 30 N é aplicada ao cabo da chave. Determine o momento dessa força em relação ao ponto  $O$ . Especifique os ângulos diretores coordenados  $\alpha$ ,  $\beta$  e  $\gamma$  do eixo do momento.



Problema 4.169

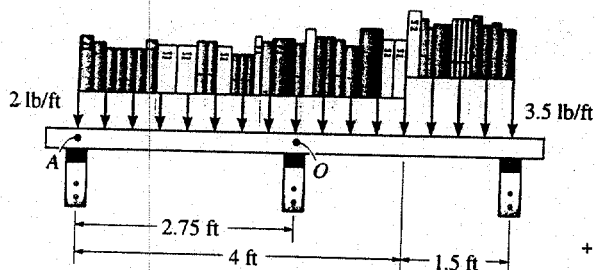
**4.170.** As forças e os momentos que são aplicados nos suportes dos dedos e do calcanhar de um esqui de neve são  $\mathbf{F}_t = \{-50\mathbf{i} + 80\mathbf{j} - 158\mathbf{k}\}$  N,  $\mathbf{M}_t = \{-6\mathbf{i} + 4\mathbf{j} + 2\mathbf{k}\}$  N·m e  $\mathbf{F}_h = \{-20\mathbf{i} + 60\mathbf{j} - 250\mathbf{k}\}$  N,  $\mathbf{M}_h = \{-20\mathbf{i} + 8\mathbf{j} + 3\mathbf{k}\}$  N·m, respectivamente. Substitua esse sistema por uma força e um



Problema 4.170

momento equivalentes que atuam no ponto  $P$ . Expresse os resultados na forma de vetores cartesianos.

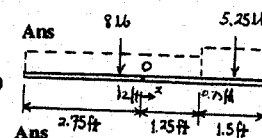
4-139. The loading on the bookshelf is distributed. Determine the magnitude of the equivalent resultant location, measured from point  $O$ .



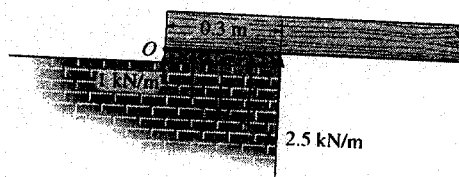
$$+\downarrow F_{RO} = \Sigma F; \quad F_{RO} = 8 + 5.25 = 13.25 = 13.2 \text{ lb } \downarrow$$

$$\zeta + M_{RO} = \Sigma M_O; \quad 13.25x = 5.25(0.75 + 1.25) - 8(2 - 1.25)$$

$$x = 0.340 \text{ ft}$$



\*4-140. The masonry support creates the loading distribution acting on the end of the beam. Simplify this load to a single resultant force and specify its location measured from point  $O$ .



**Equivalent Resultant Force :**

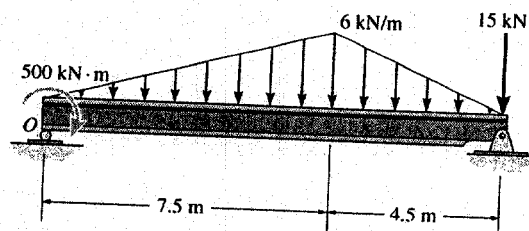
$$+\uparrow F_R = \Sigma F; \quad F_R = 0.300 + 0.225 = 0.525 \text{ kN } \uparrow \quad \text{Ans}$$

**Location of Equivalent Resultant Force :**

$$\zeta + (M_R)_O = \Sigma M_O; \quad 0.525(d) = 0.300(0.15) + 0.225(0.2)$$

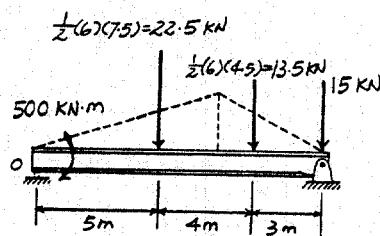
$$d = 0.171 \text{ m} \quad \text{Ans}$$

4-141. Replace the loading by an equivalent force and couple moment acting at point  $O$ .



$$+\uparrow F_R = \Sigma F; \quad F_R = -22.5 - 13.5 - 15.0 = -51.0 \text{ kN} = 51.0 \text{ kN } \downarrow \quad \text{Ans}$$

$$\zeta + M_{R_o} = \Sigma M_O; \quad M_{R_o} = -500 - 22.5(5) - 13.5(9) - 15(12) = -914 \text{ kN}\cdot\text{m} = 914 \text{ kN}\cdot\text{m} \text{ (Clockwise)} \quad \text{Ans}$$



4-142. Replace the loading by a single resultant force, and specify the location of the force on the beam measured from point  $O$ .

**Equivalent Resultant Force :**

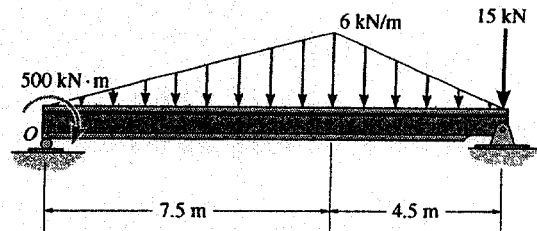
$$+\uparrow F_R = \Sigma F_y; \quad -F_R = -22.5 - 13.5 - 15$$

$$F_R = 51.0 \text{ kN} \downarrow \quad \text{Ans}$$

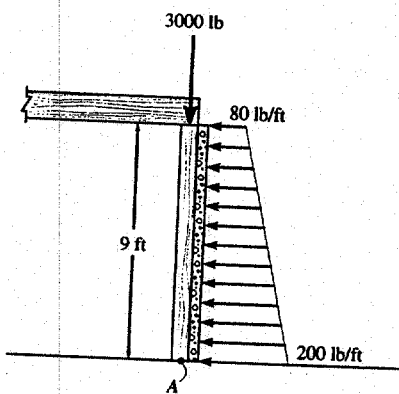
**Location of Equivalent Resultant Force :**

$$(+M_R)_O = \Sigma M_O; \quad -51.0(d) = -500 - 22.5(5) - 13.5(9) - 15(12)$$

$$d = 17.9 \text{ m} \quad \text{Ans}$$



4-143. The column is used to support the floor which exerts a force of 3000 lb on the top of the column. The effect of soil pressure along its side is distributed as shown. Replace this loading by an equivalent resultant force and specify where it acts along the column, measured from its base  $A$ .



$$\leftarrow \Sigma F_{Rx} = \Sigma F_x; \quad F_{Rx} = 720 + 540 = 1260 \text{ lb}$$

$$+\downarrow F_{Ry} = \Sigma F_y; \quad F_{Ry} = 3000 \text{ lb}$$

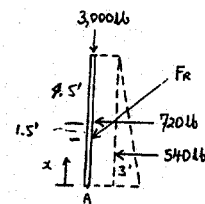
$$F_R = \sqrt{(1260)^2 + (3000)^2} = 3254 \text{ lb}$$

$$F_R = 3.25 \text{ kip} \quad \text{Ans}$$

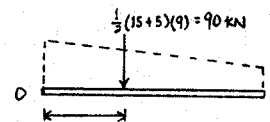
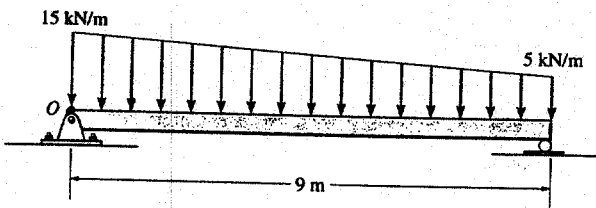
$$\theta = \tan^{-1} \left[ \frac{3000}{1260} \right] = 67.2^\circ \text{ Ans}$$

$$(+M_{RA}) = \Sigma M_A; \quad 1260x = 540(3) + 720(4.5)$$

$$x = 3.86 \text{ ft} \quad \text{Ans}$$



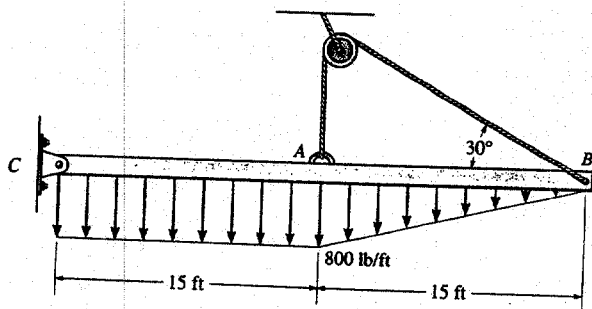
\*4-144. Replace the loading by an equivalent force and couple moment acting at point  $O$ .



$$+\downarrow F_R = \Sigma F; \quad F_R = 90 \text{ kN} \downarrow \quad \text{Ans}$$

$$\zeta (+M_{RO}) = \Sigma M_O; \quad M_{RO} = 90(3.75) = 338 \text{ kN}\cdot\text{m} \quad \text{Ans}$$

4-145. Replace the distributed loading by an equivalent resultant force, and specify its location on the beam, measured from the pin at  $C$ .

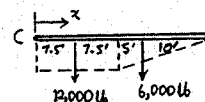


$$+\downarrow F_R = \Sigma F; \quad F_R = 12000 + 6000 = 18000 \text{ lb}$$

$$F_R = 18.0 \text{ kip} \downarrow \quad \text{Ans}$$

$$\zeta (+M_{RC}) = \Sigma M_C; \quad 18000x = 12000(7.5) + 6000(20)$$

$$x = 11.7 \text{ ft} \quad \text{Ans}$$



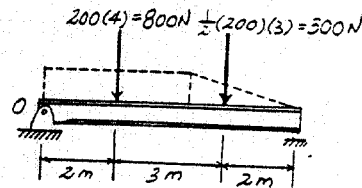
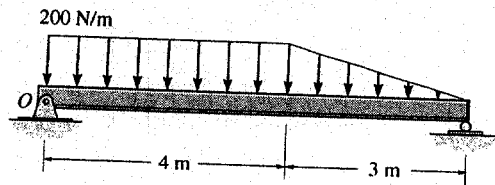


4-146. Replace the loading by an equivalent force and couple moment acting at point  $O$ .

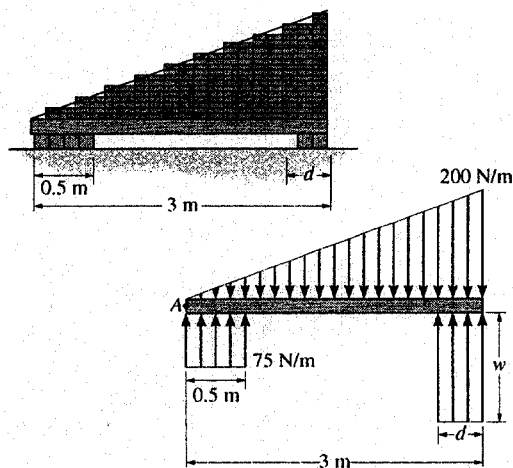
Equivalent Force and Couple Moment At Point  $O$ :

$$+\uparrow F_R = \Sigma F_y; \quad F_R = -800 - 300 \\ = -1100 \text{ N} = 1.10 \text{ kN} \downarrow \quad \text{Ans}$$

$$+ M_{R_o} = \Sigma M_o; \quad M_{R_o} = -800(2) - 300(5) \\ = -3100 \text{ N}\cdot\text{m} \\ = 3.10 \text{ kN}\cdot\text{m} \text{ (Clockwise)} \quad \text{Ans}$$



\*4-147. The bricks on top of the beam and the supports at the bottom create the distributed loading shown in the second figure. Determine the required intensity  $w$  and dimension  $d$  of the right support so that the resultant force and couple moment about point  $A$  of the system are both zero.

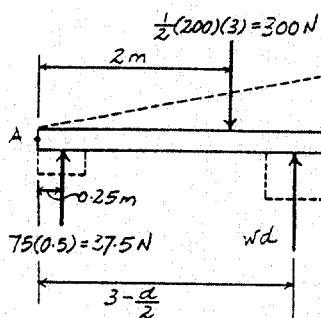


Require  $F_R = 0$ .

$$+\uparrow F_R = \Sigma F_y; \quad 0 = wd + 37.5 - 300 \\ wd = 262.5 \quad [1]$$

Require  $M_{R_A} = 0$ .

$$+\circlearrowleft M_{R_A} = \Sigma M_A; \quad 0 = 37.5(0.25) + wd\left(3 - \frac{d}{2}\right) - 300(2) \\ 3wd - \frac{wd^2}{2} = 590.625 \quad [2]$$



Solving Eqs. [1] and [2] yields

$$d = 1.50 \text{ m} \quad w = 175 \text{ N/m} \quad \text{Ans}$$

**\*4-148.** Replace the distributed loading by an equivalent resultant force and specify its location, measured from point A.

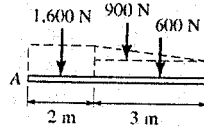
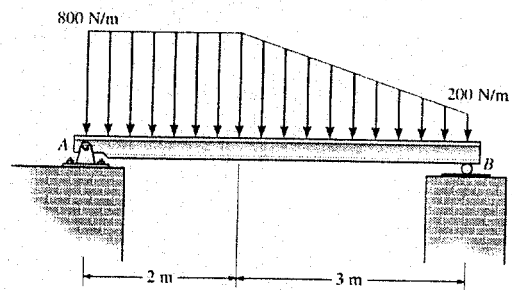
$$+\downarrow F_R = \Sigma F; \quad F_R = 1600 + 900 + 600 = 3100 \text{ N}$$

$$F_R = 3.10 \text{ kN} \downarrow \quad \text{Ans}$$

$$\curvearrowleft +M_{RA} = \Sigma M_A; \quad x(3100) = 1600(1) + 900(3) + 600(3.5)$$

$$x = 2.06 \text{ m}$$

Ans



**4-149.** The distribution of soil loading on the bottom of a building slab is shown. Replace this loading by an equivalent resultant force and specify its location, measured from point O.

$$+\uparrow F_R = \Sigma F_y; \quad F_R = 50(12) + \frac{1}{2}(250)(12)$$

$$+ \frac{1}{2}(200)(9) + 100(9)$$

$$= 3900 \text{ lb} = 3.90 \text{ kip} \uparrow$$

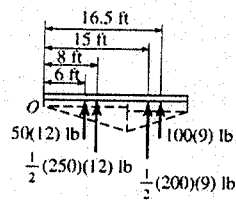
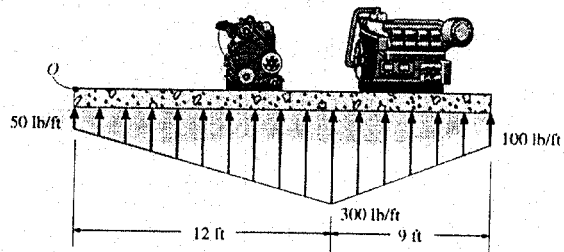
Ans

$$\curvearrowleft +M_{RO} = \Sigma M_O; \quad 3900(d) = 50(12)(6) + \frac{1}{2}(250)(12)(8)$$

$$+ \frac{1}{2}(200)(9)(15) + 100(9)(16.5)$$

$$d = 11.3 \text{ ft}$$

Ans



**4-150.** The beam is subjected to the distributed loading. Determine the length  $b$  of the uniform load and its position  $a$  on the beam such that the resultant force and couple moment acting on the beam are zero.

Require  $F_R = 0$ .

$$+\uparrow F_R = \Sigma F_y; \quad 0 = 180 - 40b$$

$$b = 4.50 \text{ ft}$$

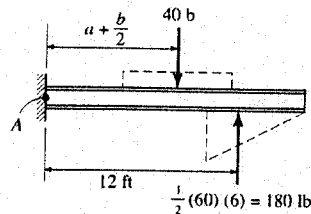
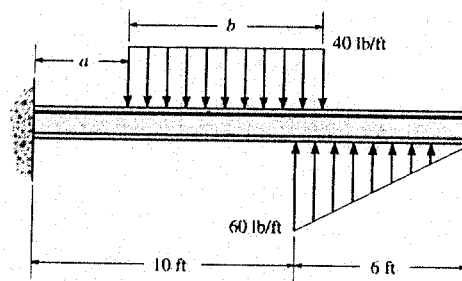
Ans

Require  $M_{RA} = 0$ . Using the result  $b = 4.50$  ft, we have

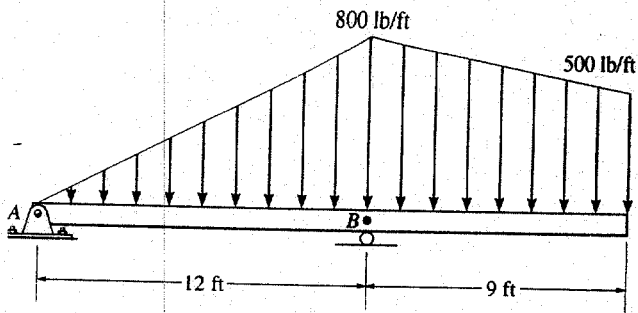
$$\curvearrowleft +M_{RA} = \Sigma M_A; \quad 0 = 180(12) - 40(4.50) \left( a + \frac{4.50}{2} \right)$$

$$a = 9.75 \text{ ft}$$

Ans



4-151. Replace the loading by an equivalent resultant force and specify its location on the beam, measured from point B.

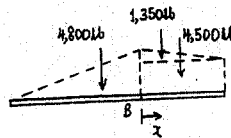


$$+\downarrow F_R = \Sigma F; \quad F_R = 4800 + 1350 + 4500 = 10\,650 \text{ lb}$$

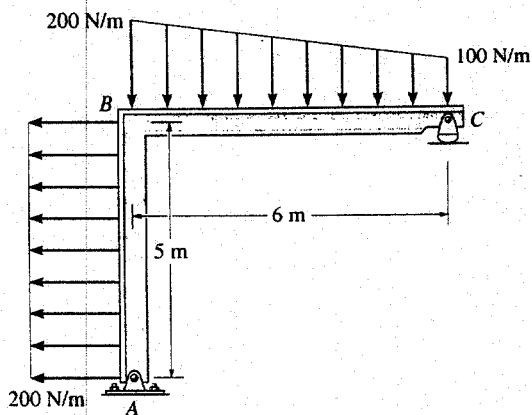
$$F_R = 10.6 \text{ kip } \downarrow \quad \text{Ans}$$

$$\zeta + M_{RB} = \Sigma M_B; \quad 10\,650x = -4800(4) + 1350(3) + 4500(4.5)$$

$$x = 0.479 \text{ ft} \quad \text{Ans}$$



\*4-152. Replace the distributed loading by an equivalent resultant force and specify where its line of action intersects member AB, measured from A.



$$\leftarrow \Sigma F_{Rx} = \Sigma F_x; \quad F_{Rx} = 1000 \text{ N}$$

$$+\downarrow F_{Ry} = \Sigma F_y; \quad F_{Ry} = 900 \text{ N}$$

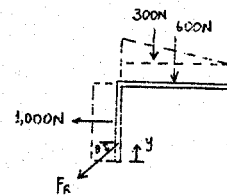
$$F_R = \sqrt{(1000)^2 + (900)^2} = 1345 \text{ N}$$

$$F_R = 1.35 \text{ kN} \quad \text{Ans}$$

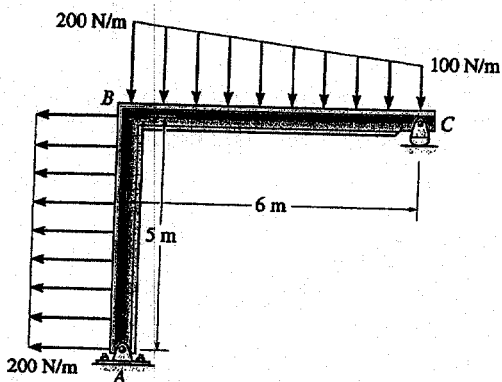
$$\theta = \tan^{-1} \left[ \frac{900}{1000} \right] = 42.0^\circ \text{ } \nearrow \text{ Ans}$$

$$\zeta + M_{RA} = \Sigma M_A; \quad 1000y = 1000(2.5) - 300(2) - 600(3)$$

$$y = 0.1 \text{ m} \quad \text{Ans}$$



4-153. Replace the distributed loading by an equivalent resultant force and specify where its line of action intersects member BC, measured from C.



$$\leftarrow \Sigma F_{Rx} = \Sigma F_x; \quad F_{Rx} = 1000 \text{ N}$$

$$+\downarrow F_{Ry} = \Sigma F_y; \quad F_{Ry} = 900 \text{ N}$$

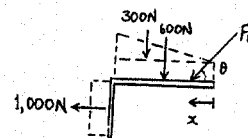
$$F_R = \sqrt{(1000)^2 + (900)^2} = 1345 \text{ N}$$

$$F_R = 1.35 \text{ kN} \quad \text{Ans}$$

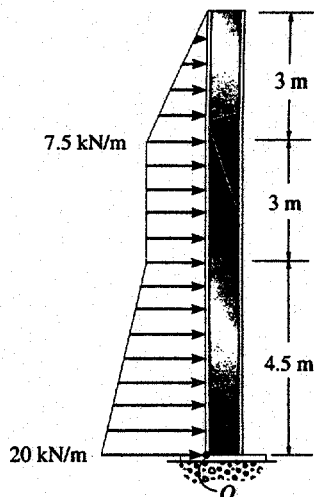
$$\theta = \tan^{-1} \left[ \frac{900}{1000} \right] = 42.0^\circ \text{ } \nearrow \text{ Ans}$$

$$\zeta + M_{RC} = \Sigma M_C; \quad 900x = 600(3) + 300(4) - 1000(2.5)$$

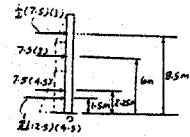
$$x = 0.556 \text{ m} \quad \text{Ans}$$



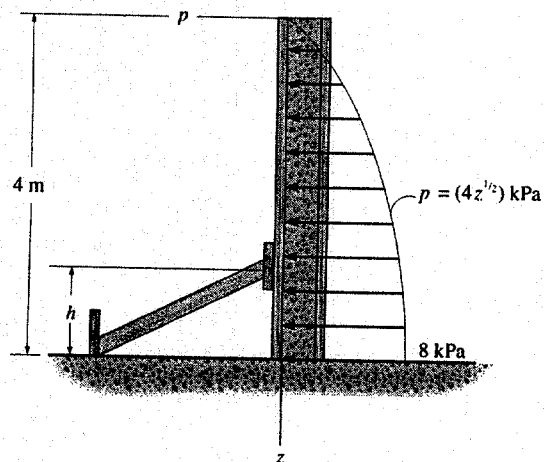
4-154. Replace the loading by an equivalent resultant force and couple moment acting at point  $O$ .



$$\begin{aligned} \rightarrow F_R = \Sigma F_x; \quad F_R &= \frac{1}{2}(12.5)(4.5) + 7.5(4.5) + 7.5(3) + \frac{1}{2}(7.5)(3) \\ &= 95.6 \text{ kN} \rightarrow \quad \text{Ans} \\ \curvearrowright +M_{R_O} = \Sigma M_O; \quad M_{R_O} &= -\frac{1}{2}(12.5)(4.5)(1.5) - 7.5(4.5)(2.25) - 7.5(3)(6) - \frac{1}{2}(7.5)(3)(8.5) \\ &= -349 \text{ kN} \cdot \text{m} = 349 \text{ kN} \cdot \text{m} \quad \text{Ans} \end{aligned}$$



4-155. Wet concrete exerts a pressure distribution along the wall of the form. Determine the resultant force of this distribution and specify the height  $h$  where the bracing strut should be placed so that it lies through the line of action of the resultant force. The wall has a width of 5 m.



**Equivalent Resultant Force :**

$$\begin{aligned} \rightarrow F_R = \Sigma F_x; \quad -F_R &= -\int_A dA = -\int_0^4 w dz \\ F_R &= \int_0^{4\text{m}} (20z^{1/2})(10^3) dz \\ &= 106.67(10^3) \text{ N} = 107 \text{ kN} \leftarrow \quad \text{Ans} \end{aligned}$$

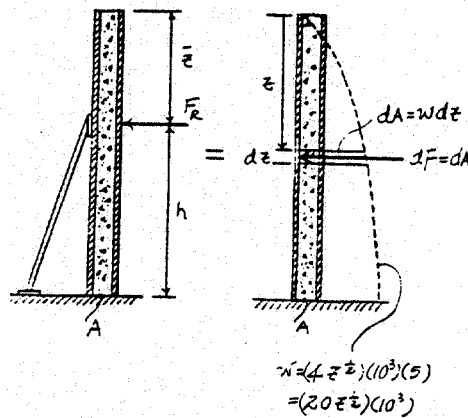
**Location of Equivalent Resultant Force :**

$$\begin{aligned} \bar{z} &= \frac{\int_A z dA}{\int_A dA} = \frac{\int_0^4 zw dz}{\int_0^4 w dz} \\ &= \frac{\int_0^{4\text{m}} z[(20z^{1/2})(10^3)] dz}{\int_0^{4\text{m}} (20z^{1/2})(10^3) dz} \\ &= \frac{\int_0^{4\text{m}} (20z^{3/2})(10^3) dz}{\int_0^{4\text{m}} (20z^{1/2})(10^3) dz} \\ &= 2.40 \text{ m} \end{aligned}$$

Thus,

$$h = 4 - \bar{z} = 4 - 2.40 = 1.60 \text{ m}$$

Ans



\*4-156. Wind has blown sand over a platform such that the intensity of the load can be approximated by the function  $w = (0.5x^3)$  N/m. Simplify this distributed loading to an equivalent resultant force and specify the magnitude and location of the force, measured from A.

$$dA = w dx$$

$$F_R = \int dA = \int_0^{10} \frac{1}{2} x^3 dx$$

$$= \left[ \frac{1}{8} x^4 \right]_0^{10}$$

$$= 1250 \text{ N}$$

$$F_R = 1.25 \text{ kN}$$

Ans

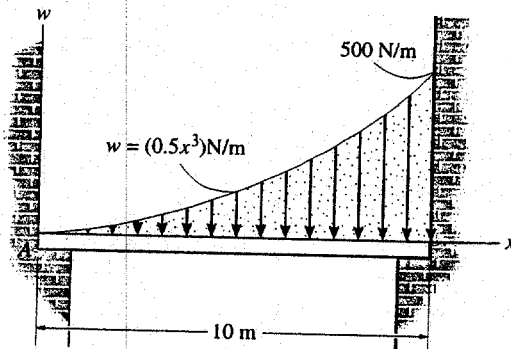
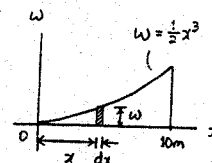
$$\int \bar{x} dA = \int_0^{10} \frac{1}{2} x^4 dx$$

$$= \left[ \frac{1}{10} x^5 \right]_0^{10}$$

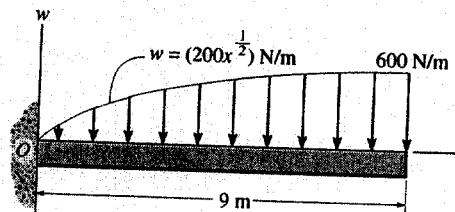
$$= 10000 \text{ N}\cdot\text{m}$$

$$\bar{x} = \frac{10000}{1250} = 8.00 \text{ m}$$

Ans



4-157. Replace the loading by an equivalent force and couple moment acting at point O.



**Equivalent Resultant Force And Moment At Point O :**

$$+\uparrow F_R = \Sigma F_y; \quad F_R = -\int_A dA = -\int_0^9 w dx$$

$$F_R = -\int_0^9 (200x^{1/2}) dx$$

$$= -3600 \text{ N} = 3.60 \text{ kN} \downarrow$$

Ans

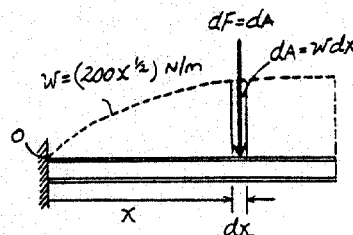
$$(+ M_{R_o} = \Sigma M_o; \quad M_{R_o} = -\int_0^9 x w dx$$

$$= -\int_0^9 x (200x^{1/2}) dx$$

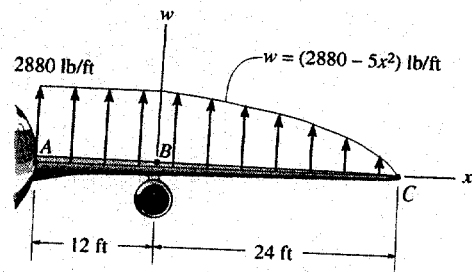
$$= -\int_0^9 (200x^{3/2}) dx$$

$$= -19440 \text{ N}\cdot\text{m}$$

$$= 19.4 \text{ kN}\cdot\text{m (Clockwise)} \quad \text{Ans}$$

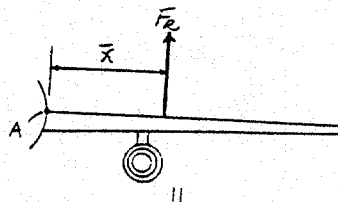


**\*4-158.** The lifting force along the wing of a jet aircraft consists of a uniform distribution along  $AB$ , and a semiparabolic distribution along  $BC$  with origin at  $B$ . Replace this loading by a single resultant force and specify its location measured from point  $A$ .



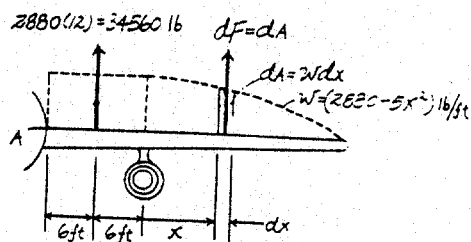
**Equivalent Resultant Force :**

$$\begin{aligned}
 + \uparrow F_R &= \Sigma F_y; & F_R &= 34560 + \int_0^{24} w dx \\
 F_R &= 34560 + \int_0^{24} (2880 - 5x^2) dx \\
 &= 80640 \text{ lb} = 80.6 \text{ kip} \uparrow & \text{Ans}
 \end{aligned}$$

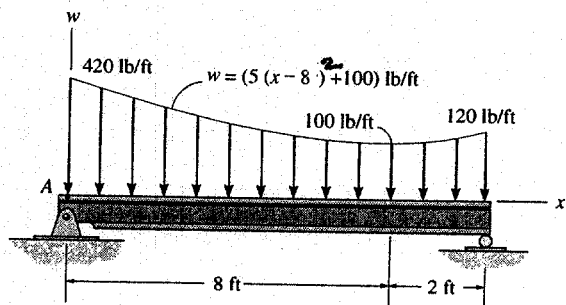


**Location of Equivalent Resultant Force :**

$$\begin{aligned}
 (+ M_{R_x} &= \Sigma M_A; \\
 80640 \bar{x} &= 34560(6) + \int_0^{24} (x+12) w dx \\
 80640 \bar{x} &= 207360 + \int_0^{24} (x+12)(2880 - 5x^2) dx \\
 80640 \bar{x} &= 207360 + \int_0^{24} (-5x^3 - 60x^2 + 2880x + 34560) dx \\
 \bar{x} &= 14.6 \text{ ft} & \text{Ans}
 \end{aligned}$$

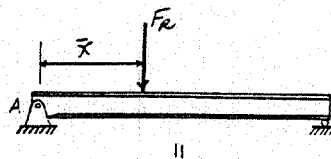


**4-159.** Determine the magnitude of the equivalent resultant force of the distributed load and specify its location on the beam measured from point  $A$ .



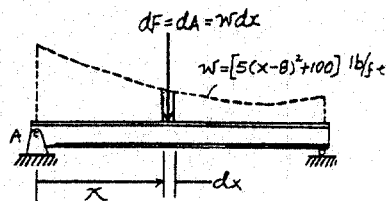
**Equivalent Resultant Force :**

$$\begin{aligned}
 + \uparrow F_R &= \Sigma F_y; & -F_R &= -\int_A dA = -\int_0^{10} w dx \\
 F_R &= \int_0^{10} [5(x-8)^2 + 100] dx \\
 &= 1866.67 \text{ lb} = 1.87 \text{ kip} \downarrow & \text{Ans}
 \end{aligned}$$

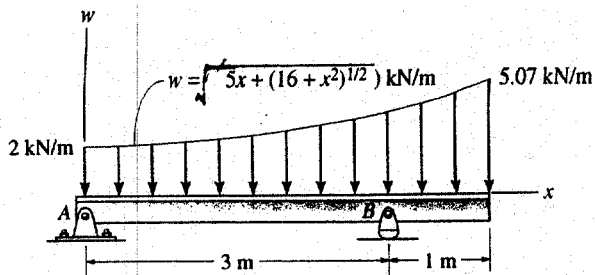


**Location of Equivalent Resultant Force :**

$$\begin{aligned}
 \bar{x} &= \frac{\int_A x dA}{\int_A dA} = \frac{\int_0^{10} x w dx}{\int_0^{10} w dx} \\
 &= \frac{\int_0^{10} x [5(x-8)^2 + 100] dx}{\int_0^{10} [5(x-8)^2 + 100] dx} \\
 &= \frac{\int_0^{10} (5x^3 - 80x^2 + 420x) dx}{\int_0^{10} [5(x-8)^2 + 100] dx} \\
 &= 3.66 \text{ ft} & \text{Ans}
 \end{aligned}$$



**\*4-160.** Determine the equivalent resultant force of the distributed loading and its location, measured from point A. Evaluate the integrals using Simpsom's rule.



$$F_R = \int w dx = \int_0^4 \sqrt{5x + (16 + x^2)^{1/2}} dx$$

$$F_R = 14.9 \text{ kN} \quad \text{Ans}$$

$$\int_0^4 \bar{x} dF = \int_0^4 (x) \sqrt{5x + (16 + x^2)^{1/2}} dx$$

$$= 33.74 \text{ kN}\cdot\text{m}$$

$$\bar{x} = \frac{33.74}{14.9} = 2.27 \text{ m} \quad \text{Ans}$$

**4-161.** Determine the coordinate direction angles  $\alpha$ ,  $\beta$ ,  $\gamma$  of  $F$ , which is applied to the end A of the pipe assembly, so that the moment of  $F$  about O is zero.

Require  $M_O = 0$ . This happens when force  $F$  is directed along line  $OA$  either from point  $O$  to  $A$  or from point  $A$  to  $O$ . The unit vectors  $u_{OA}$  and  $u_{AO}$  are

$$u_{OA} = \frac{(6-0)\mathbf{i} + (14-0)\mathbf{j} + (10-0)\mathbf{k}}{\sqrt{(6-0)^2 + (14-0)^2 + (10-0)^2}}$$

$$= 0.3293\mathbf{i} + 0.7683\mathbf{j} + 0.5488\mathbf{k}$$

Thus,

$$\alpha = \cos^{-1} 0.3293 = 70.8^\circ \quad \text{Ans}$$

$$\beta = \cos^{-1} 0.7683 = 39.8^\circ \quad \text{Ans}$$

$$\gamma = \cos^{-1} 0.5488 = 56.7^\circ \quad \text{Ans}$$

$$u_{AO} = \frac{(0-6)\mathbf{i} + (0-14)\mathbf{j} + (0-10)\mathbf{k}}{\sqrt{(0-6)^2 + (0-14)^2 + (0-10)^2}}$$

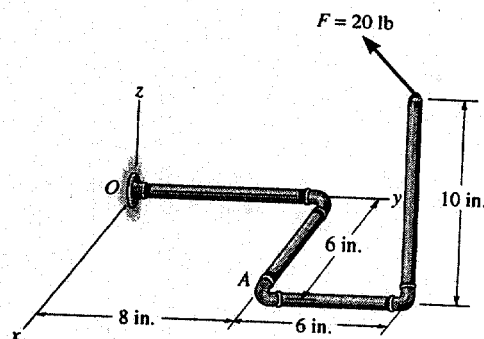
$$= -0.3293\mathbf{i} - 0.7683\mathbf{j} - 0.5488\mathbf{k}$$

Thus,

$$\alpha = \cos^{-1} (-0.3293) = 109^\circ \quad \text{Ans}$$

$$\beta = \cos^{-1} (-0.7683) = 140^\circ \quad \text{Ans}$$

$$\gamma = \cos^{-1} (-0.5488) = 123^\circ \quad \text{Ans}$$



**4-162.** Determine the moment of the force  $F$  about point  $O$ . The force has coordinate direction angles of  $\alpha = 60^\circ$ ,  $\beta = 120^\circ$ ,  $\gamma = 45^\circ$ . Express the result as a Cartesian vector.

**Position Vector And Force Vectors :**

$$r_{OA} = \{(6-0)\mathbf{i} + (14-0)\mathbf{j} + (10-0)\mathbf{k}\} \text{ in.}$$

$$= \{6\mathbf{i} + 14\mathbf{j} + 10\mathbf{k}\} \text{ in.}$$

$$F = 20(\cos 60^\circ\mathbf{i} + \cos 120^\circ\mathbf{j} + \cos 45^\circ\mathbf{k}) \text{ lb}$$

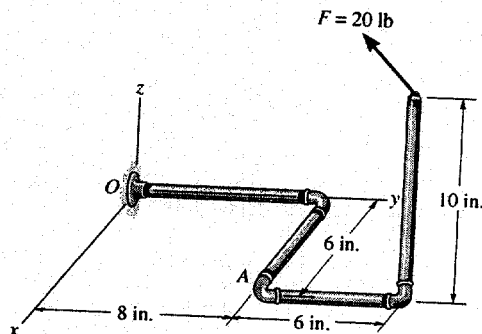
$$= \{10.0\mathbf{i} - 10.0\mathbf{j} + 14.142\mathbf{k}\} \text{ lb}$$

**Moment of Force F About Point O :** Applying Eq. 4-7, we have

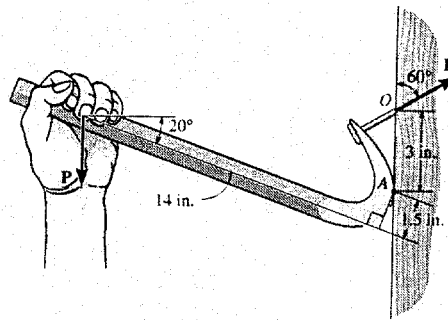
$$M_O = r_{OA} \times F$$

$$= \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 6 & 14 & 10 \\ 10.0 & -10.0 & 14.142 \end{vmatrix}$$

$$= \{298\mathbf{i} + 15.1\mathbf{j} - 200\mathbf{k}\} \text{ lb}\cdot\text{in} \quad \text{Ans}$$



4-163. If it takes a force of  $F = 125$  lb to pull the nail out, determine the smallest vertical force  $P$  that must be applied to the handle of the crowbar. *Hint:* This requires the moment of  $F$  about point  $A$  to be equal to the moment of  $P$  about  $A$ . Why?



$$\uparrow +M_F = 125(\sin 60^\circ)(3) = 324.7595 \text{ lb} \cdot \text{in.}$$

$$\uparrow +M_P = P(14 \cos 20^\circ + 1.5 \sin 20^\circ) = M_F = 324.7595 \text{ lb} \cdot \text{in.}$$

$$P = 23.8 \text{ lb}$$

Ans

\*4-164. Determine the moment of the force  $F_C$  about the door hinge at  $A$ . Express the result as a Cartesian vector.

*Position Vector And Force Vector:*

$$\mathbf{r}_{AB} = [(-0.5 - (-0.5))\mathbf{i} + [0 - (-1)]\mathbf{j} + (0 - 0)\mathbf{k}] \text{ m} = (1\mathbf{j}) \text{ m}$$

$$\mathbf{F}_C = 250 \left( \frac{[-0.5 - (-2.5)]\mathbf{i} + [0 - (-1 + 1.5 \cos 30^\circ)]\mathbf{j} + [0 - 1.5 \sin 30^\circ]\mathbf{k}}{\sqrt{[0 - (-1 + 1.5 \cos 30^\circ)]^2 + (-0.5 - (-2.5))^2 + [0 - 1.5 \sin 30^\circ]^2}} \right) \text{ N}$$

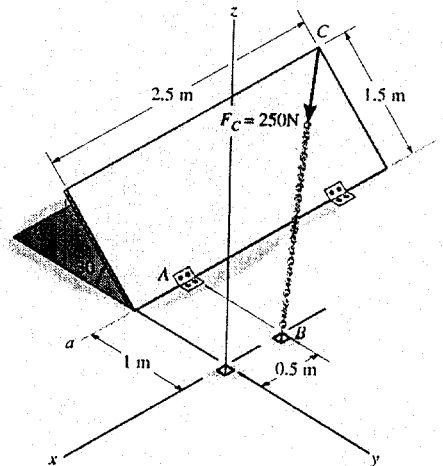
$$= (159.33\mathbf{i} + 183.15\mathbf{j} - 59.75\mathbf{k}) \text{ N}$$

*Moment of Force  $F_C$  About Point  $A$ :* Applying Eq. 4-7, we have

$$\mathbf{M}_A = \mathbf{r}_{AB} \times \mathbf{F}$$

$$= \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 0 & 1 & 0 \\ 159.33 & 183.15 & -59.75 \end{vmatrix}$$

$$= (-59.7\mathbf{i} - 159\mathbf{k}) \text{ N} \cdot \text{m} \quad \text{Ans}$$



4-165. Determine the magnitude of the moment of the force  $F_C$  about the hinged axis  $aa$  of the door.

*Position Vector And Force Vectors:*

$$\mathbf{r}_{AB} = [(-0.5 - (-0.5))\mathbf{i} + [0 - (-1)]\mathbf{j} + (0 - 0)\mathbf{k}] \text{ m} = (1\mathbf{j}) \text{ m}$$

$$\mathbf{F}_C = 250 \left( \frac{[-0.5 - (-2.5)]\mathbf{i} + [0 - (-1 + 1.5 \cos 30^\circ)]\mathbf{j} + [0 - 1.5 \sin 30^\circ]\mathbf{k}}{\sqrt{[0 - (-1 + 1.5 \cos 30^\circ)]^2 + (-0.5 - (-2.5))^2 + [0 - 1.5 \sin 30^\circ]^2}} \right) \text{ N}$$

$$= (159.33\mathbf{i} + 183.15\mathbf{j} - 59.75\mathbf{k}) \text{ N}$$

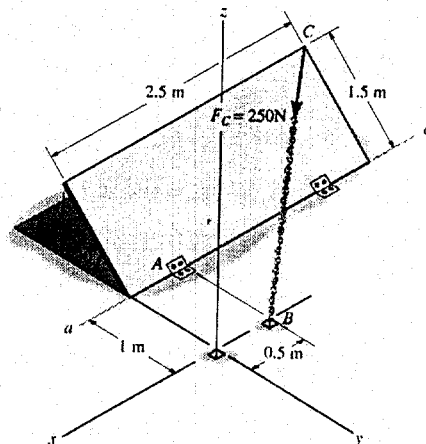
*Moment of Force  $F_C$  About  $a - a$  Axis:* The unit vector along the  $a - a$  axis is  $\mathbf{i}$ . Applying Eq. 4-11, we have

$$M_{a-a} = \mathbf{i} \cdot (\mathbf{r}_{AB} \times \mathbf{F}_C)$$

$$= \begin{vmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 159.33 & 183.15 & -59.75 \end{vmatrix}$$

$$= 1[1(-59.75) - (183.15)(0)] - 0 + 0$$

$$= -59.7 \text{ N} \cdot \text{m}$$

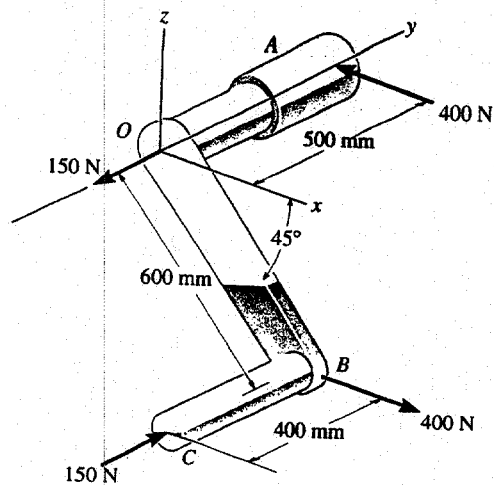


The negative sign indicates that  $M_{a-a}$  is directed toward negative  $x$  axis.

$$M_{a-a} = 59.7 \text{ N} \cdot \text{m} \quad \text{Ans}$$



4-166. Determine the resultant couple moment of the two couples that act on the assembly. Member  $OB$  lies in the  $x-z$  plane.



For the 400-N forces :

$$\begin{aligned} M_{C1} &= r_{AB} \times (400i) \\ &= \begin{vmatrix} i & j & k \\ 0.6 \cos 45^\circ & -0.5 & -0.6 \sin 45^\circ \\ 400 & 0 & 0 \end{vmatrix} \\ &= -169.7j + 200k \end{aligned}$$

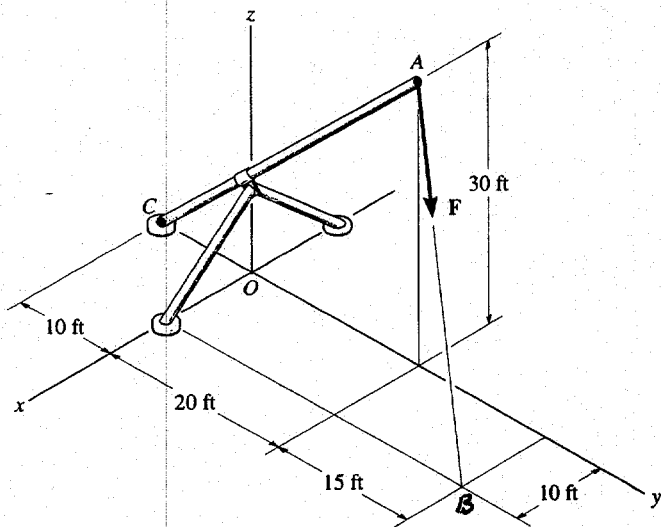
For the 150-N forces :

$$\begin{aligned} M_{C2} &= r_{OB} \times (150j) \\ &= \begin{vmatrix} i & j & k \\ 0.6 \cos 45^\circ & 0 & -0.6 \sin 45^\circ \\ 0 & 150 & 0 \end{vmatrix} \\ &= 63.6i + 63.6k \end{aligned}$$

$$M_{CR} = M_{C1} + M_{C2}$$

$$M_{CR} = \{63.6i - 170j + 264k\} \text{ N}\cdot\text{m} \quad \text{Ans}$$

4-167. Replace the force  $F$  having a magnitude of  $F = 50$  lb and acting at point  $A$  by an equivalent force and couple moment at point  $C$ .



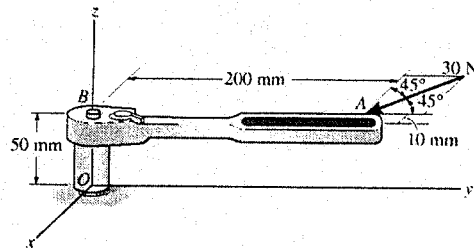
$$F_R = 50 \left[ \frac{(10i + 15j - 30k)}{\sqrt{(10)^2 + (15)^2 + (-30)^2}} \right]$$

$$F_R = \{14.3i + 21.4j - 42.9k\} \text{ lb} \quad \text{Ans}$$

$$\begin{aligned} M_{RC} &= r_{CB} \times F = \begin{vmatrix} i & j & k \\ 10 & 45 & 0 \\ 14.29 & 21.43 & -42.86 \end{vmatrix} \\ &= \{-1929i + 428.6j - 428.6k\} \text{ lb}\cdot\text{ft} \end{aligned}$$

$$M_A = \{-1.93i + 0.429j - 0.429k\} \text{ kip}\cdot\text{ft} \quad \text{Ans}$$

**\*4-168.** The horizontal 30-N force acts on the handle of the wrench. What is the magnitude of the moment of this force about the  $z$  axis?



**Position Vector And Force Vectors:**

$$\mathbf{r}_{BA} = [-0.01\mathbf{i} + 0.2\mathbf{j}] \text{ m}$$

$$\begin{aligned} \mathbf{r}_{OA} &= [(-0.01 - 0)\mathbf{i} + (0.2 - 0)\mathbf{j} + (0.05 - 0)\mathbf{k}] \text{ m} \\ &= [-0.01\mathbf{i} + 0.2\mathbf{j} + 0.05\mathbf{k}] \text{ m} \end{aligned}$$

$$\begin{aligned} \mathbf{F} &= 30(\sin 45^\circ\mathbf{i} - \cos 45^\circ\mathbf{j}) \text{ N} \\ &= [21.213\mathbf{i} - 21.213\mathbf{j}] \text{ N} \end{aligned}$$

**Moment of Force F About z Axis:** The unit vector along the  $z$  axis is  $\mathbf{k}$ . Applying Eq. 4-11, we have

$$M_z = \mathbf{k} \cdot (\mathbf{r}_{BA} \times \mathbf{F})$$

$$= \begin{vmatrix} 0 & 0 & 1 \\ -0.01 & 0.2 & 0 \\ 21.213 & -21.213 & 0 \end{vmatrix}$$

$$\begin{aligned} &= 0 - 0 + 1[(-0.01)(-21.213) - 21.213(0.2)] \\ &= -4.03 \text{ N} \cdot \text{m} \end{aligned}$$

**Ans**

Or

$$M_z = \mathbf{k} \cdot (\mathbf{r}_{OA} \times \mathbf{F})$$

$$= \begin{vmatrix} 0 & 0 & 1 \\ -0.01 & 0.2 & 0.05 \\ 21.213 & -21.213 & 0 \end{vmatrix}$$

$$= 0 - 0 + 1[(-0.01)(-21.213) - 21.213(0.2)]$$

$$= -4.03 \text{ N} \cdot \text{m}$$

**Ans**

The negative sign indicates that  $M_z$  is directed along the negative  $z$  axis.

**4-169.** The horizontal 30-N force acts on the handle of the wrench. Determine the moment of this force about point  $O$ . Specify the coordinate direction angles  $\alpha$ ,  $\beta$ ,  $\gamma$  of the moment axis.

**Position Vector And Force Vectors:**

$$\begin{aligned} \mathbf{r}_{OA} &= [(-0.01 - 0)\mathbf{i} + (0.2 - 0)\mathbf{j} + (0.05 - 0)\mathbf{k}] \text{ m} \\ &= [-0.01\mathbf{i} + 0.2\mathbf{j} + 0.05\mathbf{k}] \text{ m} \end{aligned}$$

$$\begin{aligned} \mathbf{F} &= 30(\sin 45^\circ\mathbf{i} - \cos 45^\circ\mathbf{j}) \text{ N} \\ &= [21.213\mathbf{i} - 21.213\mathbf{j}] \text{ N} \end{aligned}$$

**Moment of Force F About Point O:** Applying Eq. 4-7, we have

$$\mathbf{M}_O = \mathbf{r}_{OA} \times \mathbf{F}$$

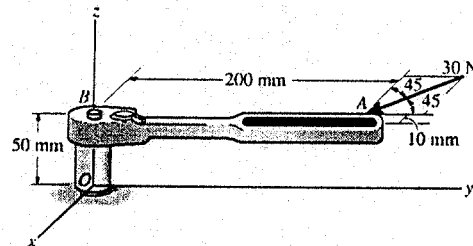
$$= \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ -0.01 & 0.2 & 0.05 \\ 21.213 & -21.213 & 0 \end{vmatrix}$$

$$= [1.061\mathbf{i} + 1.061\mathbf{j} - 4.031\mathbf{k}] \text{ N} \cdot \text{m}$$

$$= [1.06\mathbf{i} + 1.06\mathbf{j} - 4.03\mathbf{k}] \text{ N} \cdot \text{m} \quad \text{Ans}$$

The magnitude of  $\mathbf{M}_O$  is

$$M_O = \sqrt{1.061^2 + 1.061^2 + (-4.031)^2} = 4.301 \text{ N} \cdot \text{m}$$



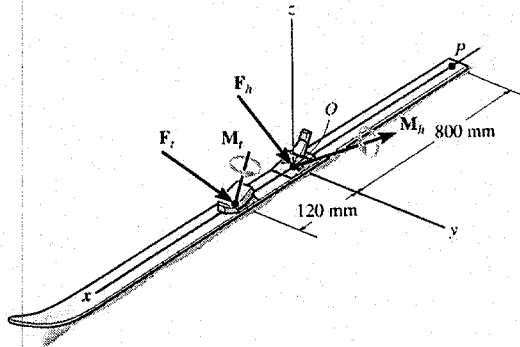
The coordinate direction angles for  $\mathbf{M}_O$  are

$$\alpha = \cos^{-1} \left( \frac{1.061}{4.301} \right) = 75.7^\circ \quad \text{Ans}$$

$$\beta = \cos^{-1} \left( \frac{1.061}{4.301} \right) = 75.7^\circ \quad \text{Ans}$$

$$\gamma = \cos^{-1} \left( \frac{-4.031}{4.301} \right) = 160^\circ \quad \text{Ans}$$

4-170. The forces and couple moments that are exerted on the toe and heel plates of a snow ski are  $\mathbf{F}_t = \{-50\mathbf{i} + 80\mathbf{j} - 158\mathbf{k}\}$  N,  $\mathbf{M}_t = \{-6\mathbf{i} + 4\mathbf{j} + 2\mathbf{k}\}$  N·m, and  $\mathbf{F}_h = \{-20\mathbf{i} + 60\mathbf{j} - 250\mathbf{k}\}$  N,  $\mathbf{M}_h = \{-20\mathbf{i} + 8\mathbf{j} + 3\mathbf{k}\}$  N·m, respectively. Replace this system by an equivalent force and couple moment acting at point  $P$ . Express the results in Cartesian vector form.



$$\mathbf{F}_R = \mathbf{F}_t + \mathbf{F}_h = \{-70\mathbf{i} + 140\mathbf{j} - 408\mathbf{k}\} \text{ N}$$

Ans

$$\mathbf{M}_{RP} = \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 0.8 & 0 & 0 \\ -20 & 60 & -250 \end{vmatrix}$$

$$+ \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ 0.92 & 0 & 0 \\ -50 & 80 & -158 \end{vmatrix} + (-6\mathbf{i} + 4\mathbf{j} + 2\mathbf{k}) + (-20\mathbf{i} + 8\mathbf{j} + 3\mathbf{k})$$

$$\mathbf{M}_{RP} = (200\mathbf{j} + 48\mathbf{k}) + (145.36\mathbf{j} + 73.6\mathbf{k})$$

$$+ (-6\mathbf{i} + 4\mathbf{j} + 2\mathbf{k}) + (-20\mathbf{i} + 8\mathbf{j} + 3\mathbf{k})$$

$$\mathbf{M}_{RP} = \{-26\mathbf{i} + 357.36\mathbf{j} + 126.6\mathbf{k}\} \text{ N} \cdot \text{m}$$

$$\mathbf{M}_{RP} = \{-26\mathbf{i} + 357\mathbf{j} + 127\mathbf{k}\} \text{ N} \cdot \text{m}$$

Ans